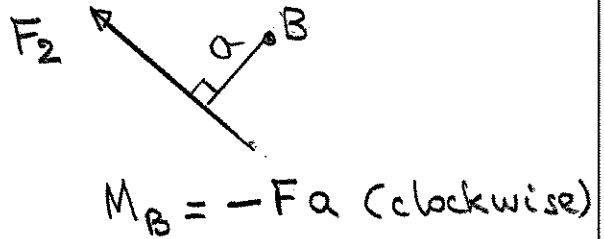
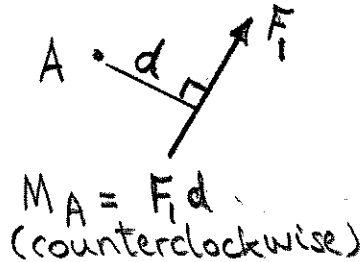


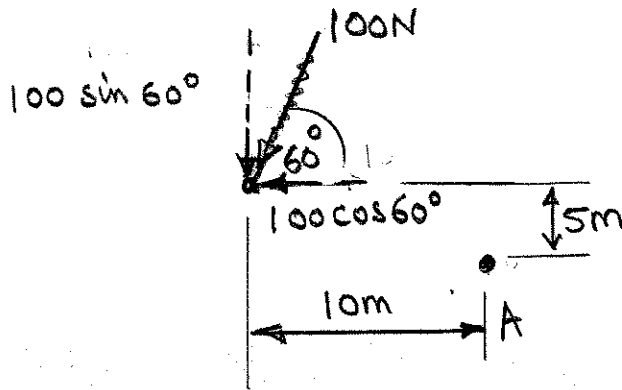
MOMENT ABOUT A POINT

MOMENT ABOUT A POINT = FORCE TIMES PERPENDICULAR DISTANCE FROM THE POINT TO THE FORCE

SIGN CONVENTION - COUNTERCLOCKWISE MOMENT IS POSITIVE (RHR).



INCLINED FORCES : RESOLVE FORCE INTO X- AND Y- COMPONENTS FIRST



$$M_A = (100 \sin 60^\circ)(10) + (100 \cos 60^\circ)(5)$$

MOMENT CAUSED BY SEVERAL FORCES
FIND RESULTANT FORCE AND POINT OF ACTION.

RESOLVE INTO X- AND Y- COMPONENTS

$$\Sigma F_x = -50 + 100 = 50 \text{ lb}$$

$$\Sigma F_y = -50 + 100 - 100 - 100 = -150 \text{ lb}$$

$$R^2 = 50^2 + 150^2 \quad R = 158.1 \text{ lb}$$

$$M_o = 50(12) - 50(12) + 100(6) = 100(12)$$

$$M_o = -600 \text{ lb-ft.}$$

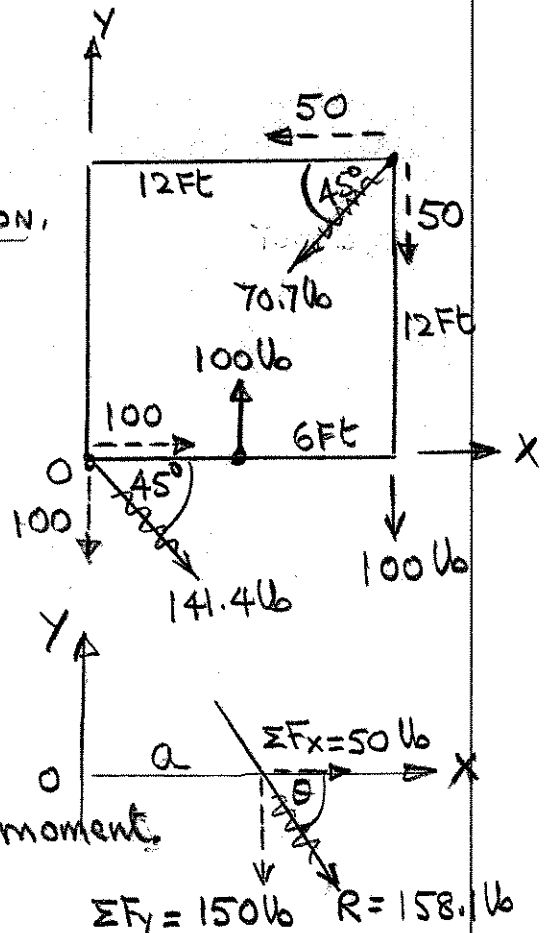
Let R pass through a point distance

'a' from O. Only vertical component causes moment.

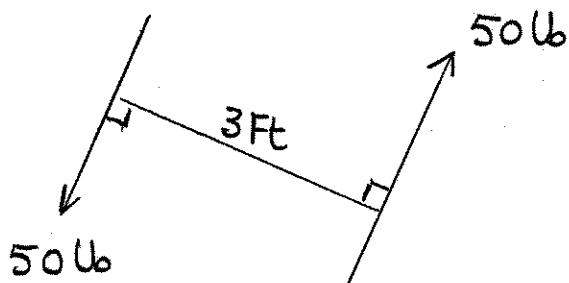
$$\text{EQUIVALENT MOMENT: } -600 = -150(a)$$

$$a = 4 \text{ ft}$$

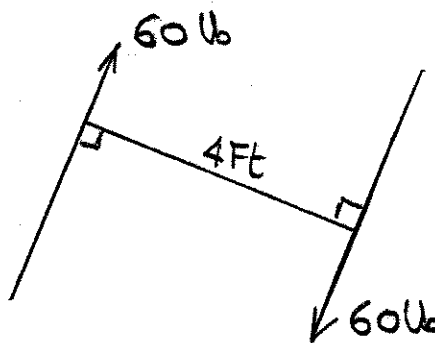
$$\tan \theta = \frac{-150}{50} = -3 \quad ; \quad \theta = -71.57^\circ \text{ OR } 288.43^\circ$$



5 COUPLE CAUSED BY EQUAL AND OPPOSITE FORCES.

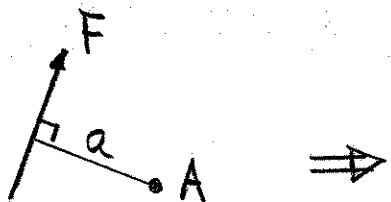


$$M = 50(3) = 150 \text{ ft-lb}$$



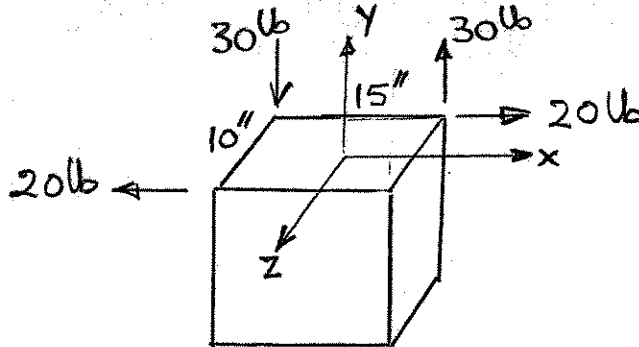
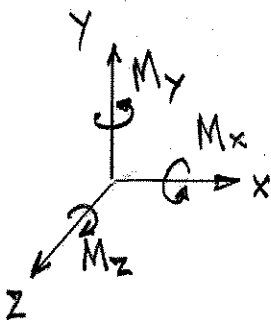
$$M = -60(4) = -240 \text{ ft-lb}$$

6. LATERAL DISPLACEMENT OF A FORCE = FORCE PLUS MOMENT



RIGHT HAND RULE - RHR

GRAB AXIS WITH RIGHT HAND SUCH THAT FINGERS CURL IN SAME DIRECTION AS MOMENT. THUMB GIVES DIRECTION OF VECTOR FOR MOMENT

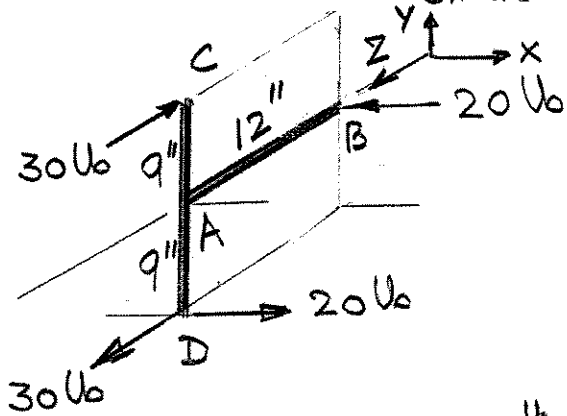


M_x IS POSITIVE
 M_y IS POSITIVE
 M_z IS NEGATIVE

Two 20-lb create $M_y = -20(10) = -200 \text{ lb-in}$

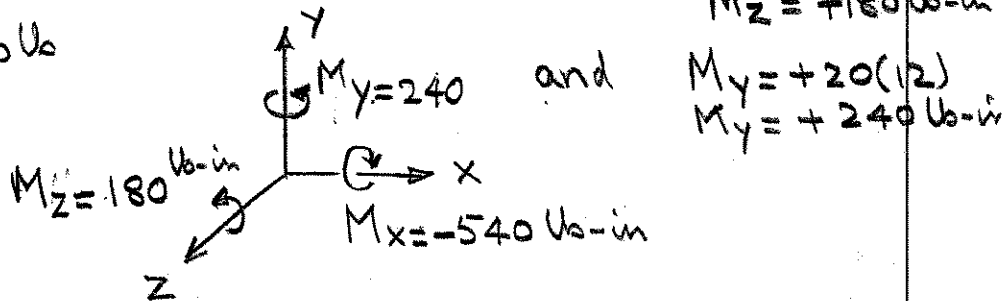
Two 30-lb $\rightarrow M_z = +30(15) = 450 \text{ lb-in}$

EXAMPLE ON COUPLES : DETERMINE COMPONENTS OF EQUIVALENT SINGLE COUPLE.

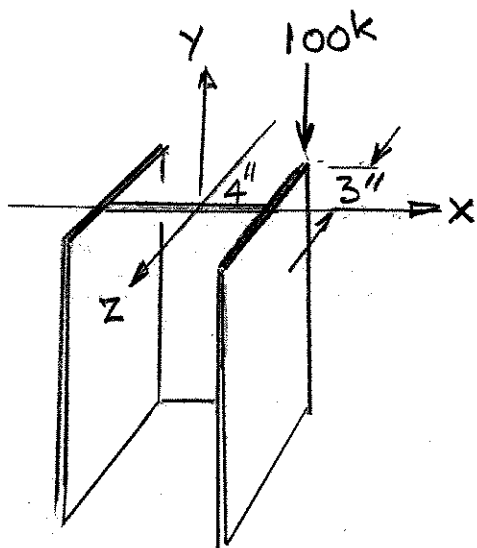


Two 30-lb create $M_x = -30(18)$
 $M_x = -540 \text{ lb-in}$
NEGATIVE SIGN BY RHR.

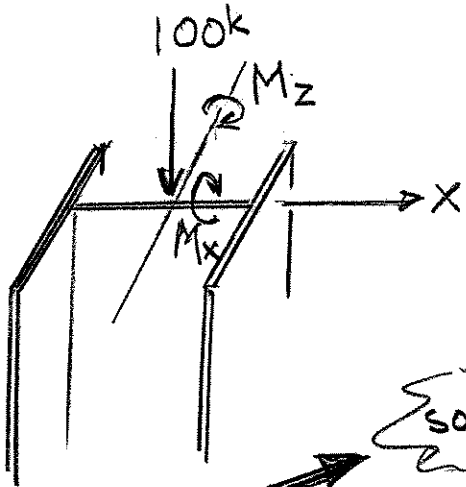
Two 20-lb create $M_z = +20(9)$
 $M_z = +180 \text{ lb-in}$



$$\underline{M} = -540 \underline{i} + 240 \underline{j} + 180 \underline{k}$$



COLUMN IS SUBJECTED TO AN ECCENTRIC LOAD OF 100k (NOT PASSING THROUGH CENTROID).
 TRANSFER LOAD TO CENTROID.



IN ADDITION TO AXIAL FORCE OF 100k, THERE EXIST MOMENTS ABOUT X- AND Z-AXIS WHICH WILL CAUSE BENDING
 $M_x = -100(3) = -300 \text{ k-in. RHR}$
 $M_z = -100(4) = -400 \text{ k-in.}$

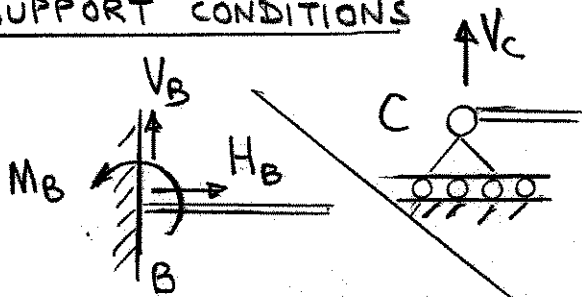
SOLVE PROBLEMS: 3.55, 3.56, 3.58, 3.62, 3.68, 3.70, 3.80, 3.82

EQUILIBRIUM - NONCONCURRENT FORCES

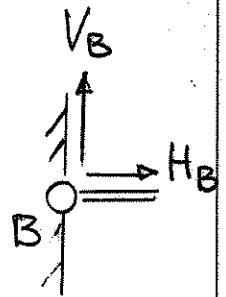
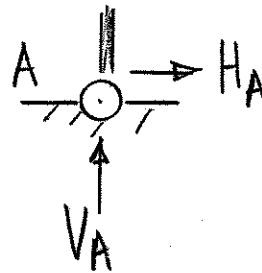
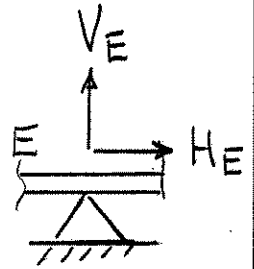
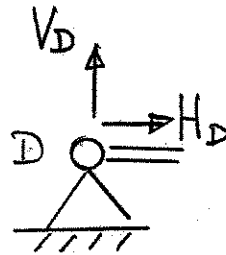
[2D] PROBLEMS : $\Sigma F_x = 0$; $\Sigma F_y = 0$; $\Sigma M = 0$

[3D] PROBLEMS : $\Sigma F_x = 0$; $\Sigma F_y = 0$; $\Sigma F_z = 0$; $\Sigma M_x = \Sigma M_y = \Sigma M_z = 0$

SUPPORT CONDITIONS



NO MOVEMENT HORIZOTALLY
 NO MOVEMENT VERTICALLY
 NO ROTATION



EXAMPLE

NOTE 3 REACTIONS.

ALWAYS APPLY $\Sigma M = 0$ FIRST
 ABOUT POINT WHERE 2 REACTIONS
 PASS.

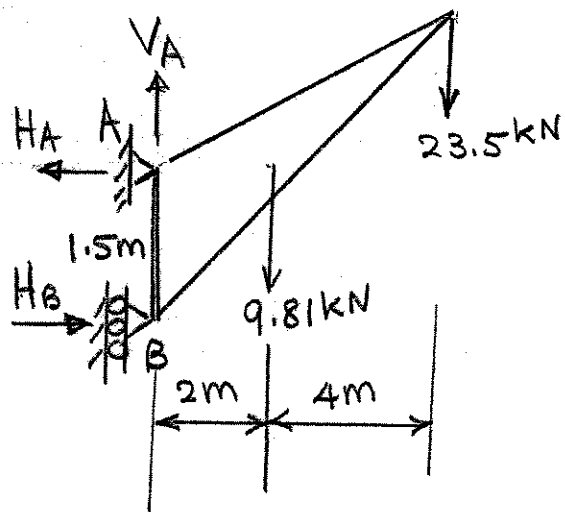
$\Sigma M_A = 0 :$

$+H_B(1.5) - 23.5(6) - 9.81(2) = 0$

$H_B = 107.1 \text{ kN}$

$\Sigma F_x = 0 : -H_A + H_B = 0 ; H_A = H_B = 107.1 \text{ kN}$

$\Sigma F_y = 0 : V_A - 23.5 - 9.81 = 0 ; V_A = 33.31 \text{ kN}$



EXAMPLE :

3 reactions • Assume V_A points upward. H_A to the left
 R_D perpendicular to the plane. NOTE 5, 12, 13 TRIANGLE
 ALWAYS APPLY $\Sigma M = 0$ FIRST.

CHOOSE A MOST CONVENIENT POINT WHERE 2 REACTIONS
 PASS. IN THIS CASE - POINT 'A'. TAKE MOMENT ABOUT POINT 'A'.
 BUT RESOLVE R_D INTO X- AND Y-COMPONENTS BEFORE DOING
 CALCULATIONS.

$\Sigma M_A = 0 :$

$-5(18) - 10(9) + R_D \sin \theta (6) + R_D \cos \theta (14) = 0$

$-90 - 90 + R_D \left(\frac{5}{13}\right)(6) + R_D \left(\frac{12}{13}\right)(14) = 0$

$R_D = 11.82 \text{ k}$

$\Sigma F_x = 0 : 5 - H_A - R_D \sin \theta = 0$

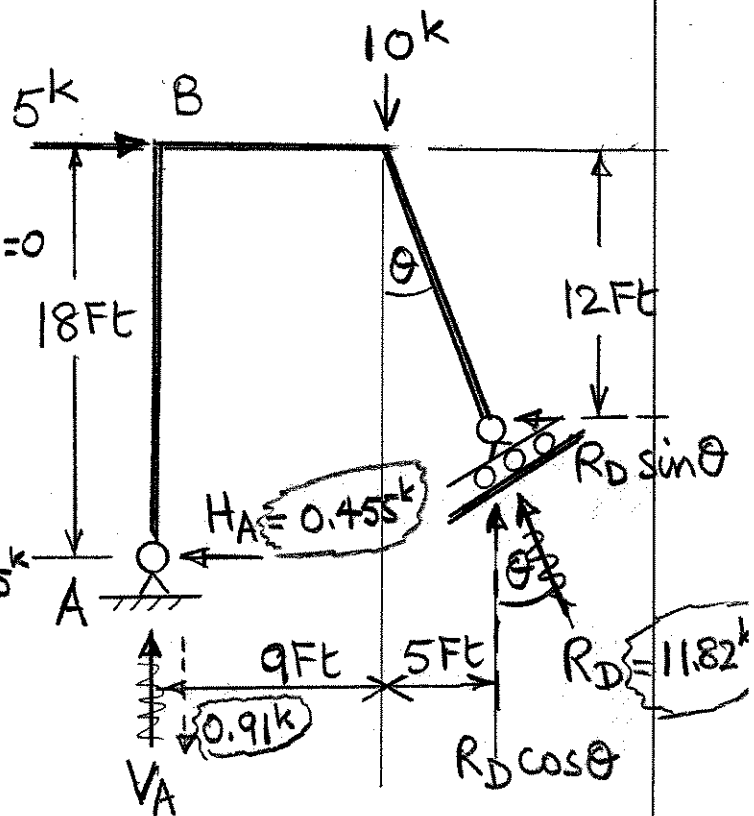
$H_A = 5 - 11.82 \left(\frac{5}{13}\right) = 0.455 \text{ k}$

$\Sigma F_y = 0 : V_A + R_D \cos \theta - 10 = 0$

$V_A = 10 - R_D \cos \theta$

$V_A = 10 - 11.82 \left(\frac{12}{13}\right) = -0.91 \text{ k}$

ORIGINAL DIRECTION CHOSEN FOR V_A WAS WRONG.
 CHANGE DIRECTION OF V_A .



SOLVE PROBLEMS: 4.4, 4.10, 4.16, 4.18(c), 4.23, 4.30,
 4.36

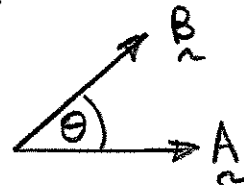
60 SHEETS FULLER 8 SQUARE
 40 SHEETS FULLER 8 SQUARE
 100 SHEETS FULLER 8 SQUARE
 200 SHEETS FULLER 8 SQUARE
 15-782
 15-781
 42-382
 42-383
 42-384
 42-385
 42-386
 Made in U.S.A.



[3D] PROBLEMS

$$\vec{A} = A_x \vec{i} + A_y \vec{j} + A_z \vec{k} \quad A = \sqrt{A_x^2 + A_y^2 + A_z^2} \quad \text{MAGNITUDE}$$

$$\vec{B} = B_x \vec{i} + B_y \vec{j} + B_z \vec{k} \quad B = \sqrt{B_x^2 + B_y^2 + B_z^2}$$



SCALAR PRODUCT: $\vec{A} \cdot \vec{B} = \vec{B} \cdot \vec{A} = AB \cos \theta$

$$\vec{A} \cdot \vec{B} = A_x B_x + A_y B_y + A_z B_z$$

CAN USE ABOVE TO CALCULATE θ .

USE OF SCALAR PRODUCT - CALCULATE PROJECTION OF A VECTOR ON A LINE.

EXAMPLE: VECTOR $\vec{A} = 30\vec{i} + 50\vec{j} + 70\vec{k}$; LINE $\vec{r} = 3\vec{i} + 2\vec{j} + 1\vec{k}$

FIRST FIND UNIT VECTOR \vec{I} in the direction of \vec{r} .

$$\text{Magnitude } r = (3^2 + 2^2 + 1^2)^{1/2} = \sqrt{14}$$

$$\vec{I} = \frac{\vec{r}}{r} = \frac{3}{\sqrt{14}} \vec{i} + \frac{2}{\sqrt{14}} \vec{j} + \frac{1}{\sqrt{14}} \vec{k}$$

PROJECTION OF \vec{A} ON \vec{r} IS GIVEN BY $\vec{A} \cdot \vec{I} = A_x I_x + A_y I_y + A_z I_z$

$$\vec{A} \cdot \vec{I} = 30\left(\frac{3}{\sqrt{14}}\right) + 50\left(\frac{2}{\sqrt{14}}\right) + 70\left(\frac{1}{\sqrt{14}}\right) = 69.49$$

VECTOR PRODUCT - NOT COMMUTATIVE $\vec{A} \times \vec{B} \neq \vec{B} \times \vec{A}$

MOMENT ABOUT A POINT $\vec{M} = \vec{r} \times \vec{F}$

\vec{r} IS A VECTOR FROM THE POINT TO THE FORCE.

EXAMPLE: FORCE $\vec{F} = 10\vec{i} - 10\vec{j} + 20\vec{k}$ passes through point P(4, 3, -2)

CALCULATE MOMENT CAUSED BY \vec{F} ABOUT POINT A(2, 3, -1)

SOLUTION: $\vec{r} = (4-2)\vec{i} + (3-3)\vec{j} + (-2+1)\vec{k} = 2\vec{i} - 1\vec{j} - 1\vec{k}$.

$$\vec{M}_A = \vec{r} \times \vec{F} = \begin{vmatrix} \vec{i} & \vec{j} & \vec{k} \\ 2 & -1 & -1 \\ 10 & -10 & 20 \end{vmatrix} = M_x \vec{i} + M_y \vec{j} + M_z \vec{k}$$

$$M_x = yF_z - zF_y = (-1)(20) - (-1)(-10) = -30 \text{ k-Ft}$$

$$M_y = zF_x - xF_z = (-1)(10) - (2)(20) = -50 \text{ k-Ft}$$

$$M_z = xF_y - yF_x = (2)(-10) - (-1)(10) = -10 \text{ k-Ft}$$

$$\vec{M}_A = -30\vec{i} - 50\vec{j} - 10\vec{k} \text{ k-Ft}$$

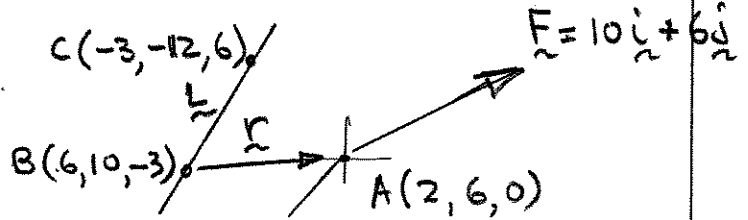
MOMENT ABOUT AN AXIS (A LINE). - 3 STEPS INVOLVED

FORCE $\vec{F} = 10\vec{i} + 6\vec{j}$ PASSES THROUGH POINT A(2, 6, 0)

FIND MOMENT CAUSED BY \vec{F} ABOUT A LINE PASSING THROUGH B(6, 10, -3) AND C(-3, -12, 6)

STEP 1: CALCULATE MOMENT ABOUT A POINT ON THE LINE.

CHOOSE POINT B.
 \vec{r} IS VECTOR FROM POINT B TO THE FORCE.



$$\vec{r} = (2-6)\vec{i} + (6-10)\vec{j} + (0+3)\vec{k} ; \vec{r} = -4\vec{i} - 4\vec{j} + 3\vec{k}$$

$$M_x = yF_z - zF_y = (-4)(0) - (-3)(6) = -18$$

$$M_y = zF_x - xF_z = (3)(10) - (-4)(0) = 30$$

$$M_z = xF_y - yF_x = (-4)(6) - (-4)(10) = 16 \quad \vec{M}_B = -18\vec{i} + 30\vec{j} + 16\vec{k}$$

STEP 2: FIND UNIT VECTOR ON AXIS

LET \vec{L} BE VECTOR FROM B TO A.

$$\vec{L} = (-3-6)\vec{i} + (-12-10)\vec{j} + (6+3)\vec{k}$$

$$\vec{L} = -9\vec{i} - 22\vec{j} + 9\vec{k}$$

$$L = \sqrt{9^2 + 22^2 + 9^2} = 25.42$$

$$\text{UNIT VECTOR } \vec{I} = \frac{\vec{L}}{L} = \frac{-9}{25.42}\vec{i} - \frac{22}{25.42}\vec{j} + \frac{9}{25.42}\vec{k}$$

$$\vec{I} = -0.354\vec{i} - 0.866\vec{j} + 0.354\vec{k}$$

STEP 3: PROJECT \vec{M}_B ON \vec{I} .

$$\vec{M} = \vec{M}_B \cdot \vec{I} = M_x I_x + M_y I_y + M_z I_z$$

$$M = (-18)(-0.354) + (30)(-0.866) + (16)(0.354) = 13.98 \text{ N-m}$$

OR USE DETERMINANT $M = \vec{I} \cdot (\vec{r} \times \vec{F}) =$

$$M = -0.354 \begin{vmatrix} 4 & 3 \\ 6 & 0 \end{vmatrix} + (-0.866) \begin{vmatrix} -4 & 3 \\ 10 & 0 \end{vmatrix} + 0.354 \begin{vmatrix} -4 & -4 \\ 10 & 6 \end{vmatrix} = 13.98$$

EXAMPLE: Ball joint at A.
Calculate T_D and T_C

REMEMBER TIP MINUS TOE.

FOR T_D : $dx = -15$; $dy = 5$; $dz = 8+5=13$
 $d^2 = dx^2 + dy^2 + dz^2 = 15^2 + 5^2 + 13^2$
 $d = 20.47$

$$\vec{T}_D = T_D \left[\frac{dx}{d} \vec{i} + \frac{dy}{d} \vec{j} + \frac{dz}{d} \vec{k} \right]$$

$$\vec{T}_D = \frac{T_D}{20.47} (-15\vec{i} + 5\vec{j} + 13\vec{k})$$

$$\vec{T}_D = (-0.733\vec{i} + 0.244\vec{j} + 0.635\vec{k}) T_D$$

Similarly $\vec{T}_C = T_C \frac{(-15\vec{i} + 9\vec{j} + 5\vec{k})}{\sqrt{15^2 + 9^2 + 5^2}} = T_C (-0.824\vec{i} + 0.495\vec{j} + 0.275\vec{k})$

Force $\vec{F} = -500\vec{j}$

NOTE: Force T_A passes through point A - causes no moment at pt A.

$\Sigma M_A = 0$: \vec{r}_{AB} is vector from point A to point B.

$$\vec{r}_{AB} = 15\vec{i} + 5\vec{j} - 5\vec{k}$$

Due to T_C : $M_x = yF_z - zF_y = 5(0.275T_C) - (-5)(0.495T_C) = 3.85T_C$

$$M_y = zF_x - xF_z = -5(-0.824) - 15(0.275) = 0$$

$$M_z = xF_y - yF_x = (15)(0.495) - (5)(-0.824) = 11.55T_C$$

Due to T_D : $M_x = 5(0.635T_D) - (-5)(0.244T_D) = 4.395T_D$

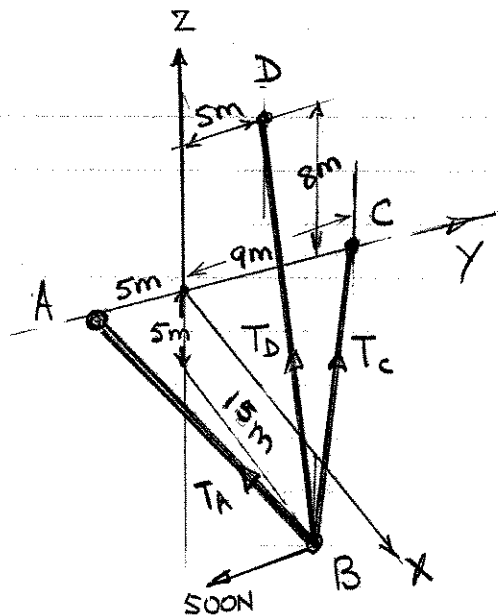
$$M_y = -5(-0.733T_D) - 15(0.635T_D) = -5.865T_D$$

$$M_z = 15(0.244T_D) - 5(-0.733T_D) = 7.325T_D$$

Due to 500 N $M_x = 0 - (-5)(-500) = -2500$

$$M_y = 0$$

$$M_z = 15(-500) = -7500$$



$$\Sigma M_x = 0: \quad 3.85T_c + 4.395T_D - 2500 = 0 \quad (1)$$

$$\Sigma M_y = 0 \quad 0 - 5.865T_D = 0 \quad (2)$$

$$\Sigma M_z = 0 \quad 11.55T_c + 7.325T_D - 7500 = 0 \quad (3)$$

$$\text{From (2)} \quad T_D = 0$$

$$\text{From (1)} \quad T_c = 649 \text{ N}$$

$$\text{From (3)} \quad T_c = 649 \text{ N}$$

SOLVE PROBLEMS 4.64, 4.70, 4.75, 4.92, 4.102

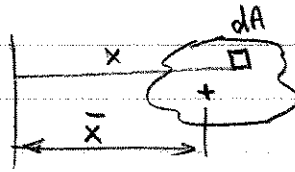
CHAPTER 5 - CENTROIDS

1. Moment caused by $dA = x \cdot dA$

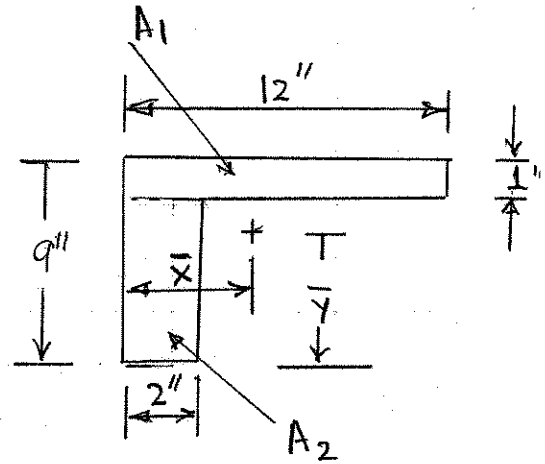
$$\text{Sum} = \sum x dA = (\sum dA) \bar{x}$$

$$\bar{x} = \frac{\sum x dA}{\sum dA} = \frac{\sum x_i A_i}{A_i}$$

$$\bar{y} = \frac{\sum A_i y_i}{A_i}$$



Area	A (in^2)	x (in)	y (in)	Ax (in^3)	Ay
A_1	12	6	8.5	72	102
A_2	16	1	4	16	64
	$\Sigma A = 28$			$\Sigma Ax = 88$	$\Sigma Ay = 166$



$$\bar{x} = \frac{\Sigma Ax}{\Sigma A} = \frac{88}{28} = 3.143''$$

$$\bar{y} = \frac{\Sigma Ay}{\Sigma A} = \frac{166}{28} = 5.929''$$

1. Centroid of a triangle

2. CENTROID BY INTEGRATION

$$dA = y dx$$

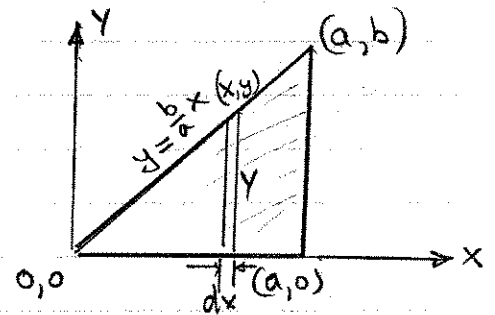
$$\bar{x} = \frac{\int x dA}{\int dA} = \frac{\int x y dx}{\int y dx}$$

$$\int x y dx = \int_0^a x \left(\frac{b}{a}x\right) dx$$

$$= \frac{b}{a} \left[\frac{x^3}{3} \right]_0^a = \frac{b}{a} \frac{a^3}{3}$$

$$\int y dx = \int_0^a \frac{b}{a} x dx = \frac{b}{a} \left[\frac{x^2}{2} \right]_0^a = \frac{ab}{2}$$

$$\therefore \bar{x} = \frac{\frac{b}{a} \frac{a^3}{3}}{\frac{ab}{2}} = \frac{2a}{3}$$



DO

3. SAMPLE PROBLEM 5.10 P251

FORCES ON CONCRETE DAM.

WORK WITH 1-FT-THICK SECTION

$W_c = 150 \text{ lb/ft}^3$ NOTE UNITS

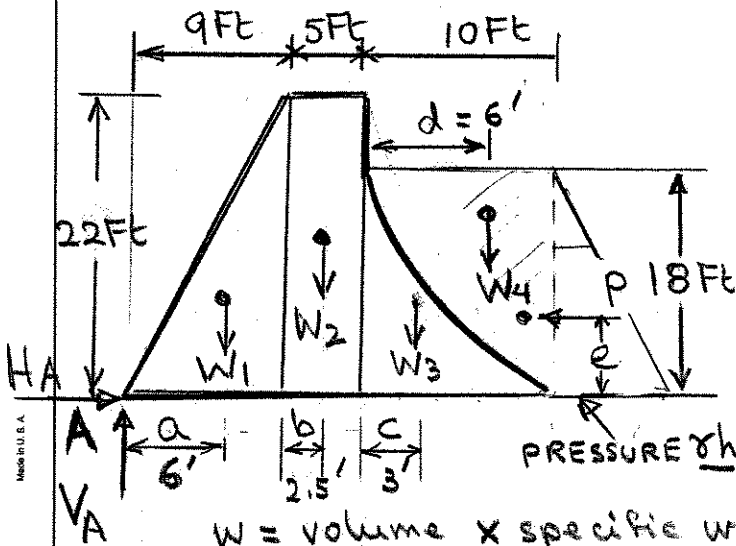
$\gamma_w = 62.4 \text{ lb/ft}^3$ SPECIFIC WEIGHT

$W_1 = W_c$ OF TRIANGULAR SECTION

$W_2 = W_c$ OF RECTANGULAR SECTION

$W_3 = W_c$ OF PARABOLIC SECTION

$W_4 = W_w$ OF WATER



VOLUME = AREA X THICKNESS

$W = \text{volume} \times \text{specific weight}$

$W_1 = \frac{1}{2}(9)(22)(1) \times 150 = 14,850 \text{ lb}$

$W_2 = (5)(22)(1) \times 150 = 16,500 \text{ lb}$

$W_3 = (\frac{1}{3})(10)(18)(1) \times 150 = 9,000 \text{ lb}$

$W_4 = \frac{2}{3}(10 \times 18 \times 1) 62.4 = 7488$
 $P = \frac{\gamma h}{2}(h \times 1) = \frac{1}{2}(62.4)(18)$
 $P = 10,109 \text{ lb}$

Location of W_1 : $a = \frac{2}{3}(9) = 6 \text{ ft}$ TRIANGLE

Location of W_2 : $b = \frac{1}{2}(5) = 2.5 \text{ ft}$ RECTANGLE

Location of W_3 : $c = \frac{3h}{10} = \frac{3}{10}(10) = 3 \text{ ft}$ PARABOLA P 225

Location of W_4 : $d = \frac{3h}{5} = \frac{3}{5}(10) = 6 \text{ ft}$ PARABOLA P 225

Location of P : $e = \frac{1}{3}(18) = 6 \text{ ft}$ TRIANGLE.

ASSUME DAM WILL ROTATE ABOUT POINT A. (LIFTING)

$\Sigma F_x = 0$: $H_A - P = 0$; $H_A = P = 10,109 \text{ lb}$.

$\Sigma F_y = 0$: $V_A - W_1 - W_2 - W_3 - W_4 = 0$; $V_A = W_1 + W_2 + W_3 + W_4 = 47836 \text{ lb}$

OVERTURNING MOMENT: $P e = 10,109(6) = 60,654 \text{ lb-ft}$.

RESTORING MOMENT: $W_1 a + W_2(9+b) + W_3(9+5+c) + W_4(9+5+d)$
 $= 14850(6) + 16,500(11.5) + 9,000(17) + 7488(20)$
 $= 581,610 \text{ lb-ft}$.

RESTORING MOMENT > OVERTURNING MOMENT O.K.